

Novel Integrated Management System Design of Electric Motorcycles

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Abstract— In view of the need for effective usage of the battery energy of the electric motorcycle, we propose an integrated management system, which not only accomplishes the objective of robust velocity tracking but also efficiently utilizes the stored energy of the battery. This integrated management system is divided into three major subsystems, including the power-saving controller, energy supervisor and protection subsystem. The concept of intelligent decision is applied such that the redundant energy loss can be avoided. At the end, simulation and experimental results will be given to demonstrate the feasibility and validity of the proposed integrated management system.

Keywords: Electric motorcycles, integrated management systems, DSP-based control.

1. INTRODUCTION

The major bottleneck in popularizing the electrical motorcycle (EM) is exactly the energy density of the battery. Although the battery technology developed in the world is gradually improved, the energy density is still not sufficient enough for mobile vehicles and it makes the user afraid of exhausting all of the battery energy before the arrival at the destination. Hence, an integrated management system with overall system dynamics, modeling, controller with power-saving strategy, energy supervisor with energy reuse, and hardware protections for the EM is of great urgency.

To concern traction control of vehicles, Kimbrough [1] has presented a method to enhance the stability and traction of a road vehicle during acceleration or braking. However, the knowledge of the friction coefficient and the slip ratio is the most dominant key point of the traction control. Therefore, Germann *et al.* [2] have proposed a friction monitoring system that utilizes recursive least squares estimation. Moreover, Sado *et al.* [3] have proposed a traction control system for the electric vehicle that includes an optimal slip control scheme with a driving-force observer to detect the road friction coefficient and a fixed-trace algorithm to estimate the slope of the slip ratio curve. As to the overall system analysis and controller design of the EM, Lin and Fu [4] have established a suitable dynamic model and used the feedback linearization for the velocity tracking control. With regard to the energy management system design, Pavlat *et al.* [5] proposed an Energy Management System (EMS) to monitor and record the State of Charge (SOC) of the battery,

vehicle performance, power consumption of the key components, and user's actions. And, it is implemented by a microprocessor-based electronic module and then modifies the inefficient energy use to extend the usable driving range and the life of the battery pack. Moreover, Jalil *et al.* [6] have accomplished a rule-based energy management strategy for a series of hybrid vehicles.

In this paper, we propose a novel integrated management system for the EM and the major viewpoint is on the highly efficient energy usage. Moreover, this management system is realized by a TI DSP F240 EVM platform and a prototyping EM to verify and illustrate the feasibility and validity of the integrated management system.

2. MODEL ANALYSIS

By the analysis of the EM, we have investigated the relations between the input and output, dynamic or static behaviors of each part of the EM system in detail and the block diagram of the EM can be established as shown in Fig.1. Afterwards, the overall dynamic model of the EM system is derived as follows:

$$\dot{\omega}_e = -a\omega_e - bN_1\mu(\lambda) + cV_c - d\tau_b \quad (1)$$

$$\dot{v} = \frac{N_1\mu(\lambda)}{M} - \frac{c_x v^2}{M} - \sigma g \cos \alpha - g \sin \alpha \quad (2)$$

where

$$a = \frac{B_{eq}R_e + K_t K_e}{J_{eq}R_e}, \quad b = \frac{rn_1 n_2}{J_{eq}}, \quad c = \frac{K_t}{J_{eq}R_e}, \quad d = \frac{n_1 n_2}{J_{eq}},$$

$$J_{eq} = J + n_1^2 n_2^2 J_r, \quad B_{eq} = B + n_1^2 n_2^2 B_r + \dot{n}_1 n_1 n_2^2 J_r$$

where ω_e is the motor velocity, N_1 is the normal force of the driving wheel, $\mu(\lambda)$ is the road friction coefficient with λ being slip ratio, V_c is the input voltage, τ_b is the braking torque, v is the vehicle velocity, M is the total mass of the EM, including the rider's weight, c_x is the coefficient of aerodynamic resistance, σ is the coefficient of rolling resistance, α is the angle of inclination, R_e is the motor resistance, K_t is the torque coefficient, K_e is the electrical coefficient, r is the radius of the wheel, n_1 is the transmission ratio of the continuous variable transmission (CVT), n_2 is the gearbox with a fixed transmission ratio, J is the moment inertia of the motor, B is the damping coefficient of the motor, J_r is the moment inertia of the driving wheel, and B_r

is the damping coefficient of the driving wheel. The motor inductance L_e is neglected here. Note that Eq. (1) describes the dynamics of the brushless DC motor (BLDCM). And Eq. (2) gives the dynamics of the vehicle velocity with the outer environmental effects such as the resistances, climbing angle, and the slip phenomenon. And, it is derived according to Newton's law.

3. MANAGEMENT SYSTEM DESIGN

The proposed management system of EM is shown in Fig. 2, including the power-saving controller, energy management, and hardware protection subsystem.

3.1 Power-Saving Controller Design

The architecture diagram of the power-saving controller is illustrated in Fig. 3, including a sliding-mode and a fuzzy controller. In order to attain a fast tracking response, we adopt the sliding-mode control to achieve that. And, in order to realize the power-saving concept, we use the fuzzy control to modify the output voltage of the sliding-mode controller based on the instantaneous information of slip ratio.

Fuzzy controller: The fuzzy controller is one SISO subsystem, where the main idea is "The larger the slip ratio, the smaller the fuzzy weighting". We utilize the center of gravity method to transfer the fuzzy sets to the desired crisp value. The membership functions of slip ratio λ and fuzzy weighting k_{fu} are shown in Fig 4, and the fuzzy rule is listed in Table I.

Sliding-mode controller: The sliding-mode controller developed is to perform the velocity of the EM. Our goal here is to design a robust controller such that the zero convergence of the tracking error as time tends to infinity can be guaranteed, i.e., the tracking error $e = v - v_d$ goes to zero asymptotically. Substituting Eq. (1) into Eq. (2) yields:

$$\dot{v} = \frac{1}{Mb} (ck_{fu}V_c - \dot{\omega}_e - a\omega_e - d\tau_b) - \frac{c_x v^2}{M} - \sigma g \cos \alpha - g \sin \alpha,$$

where we have considered the modified voltage by the fuzzy controller. The sliding surface s is defined as $s = v - v_d (= e)$ such that:

$$\dot{s} = \frac{1}{Mb} (ck_{fu}V_c - \dot{\omega}_e - a\omega_e) - \frac{c_x v^2}{M} - \sigma g \cos \alpha - g \sin \alpha - \dot{v}_d.$$

The control law is designed as follows:

$$V_c = \frac{1}{c} (a\omega_e + \hat{\omega}_e) + \frac{Mb}{c} \left[\frac{c_x v^2}{M} + g \sin \hat{\alpha} + \sigma g \cos \hat{\alpha} + \dot{v}_d - K_a \operatorname{sgn}(s) - K_p s \right], \quad (3)$$

where the constant $K_p > 0$, the adaptive law of K_a is $\dot{K}_a = k_{fu}|s|$, the nominal value of α is $\hat{\alpha}$, and the measured data of $\dot{\omega}_e$ is $\hat{\omega}_e$. Thus, the dynamics of \dot{s} becomes as follows:

$$\dot{s} = \tilde{D} - K_a k_{fu} \operatorname{sgn}(s) - K_p k_{fu} s, \quad (4)$$

where the disturbance \tilde{D} is considered as:

$$\begin{aligned} \tilde{D} = & \frac{1}{Mb} [a\omega(k_{fu} - 1) + (k_{fu} \hat{\omega}_e - \omega_e)] + \left[\frac{c_x}{M} v^2 (k_{fu} - 1) \right. \\ & + g(k_{fu} \sin \hat{\alpha} - \sin \alpha) + \sigma g(k_{fu} \cos \hat{\alpha} - \cos \alpha) \\ & \left. + \dot{v}_d (k_{fu} - 1) \right]. \end{aligned}$$

Stability Analysis: We choose the Lyapunov-like function as:

$$V = \frac{1}{2} s^2 + \frac{1}{2} (K_a - K_a^*)^2$$

whose time derivative can be evaluated as:

$$\dot{V} \leq -K_p k_{fu} s^2 - K_a^* k_{fu} |s| + |\tilde{D}| |s|.$$

Thus, if $K_a^* k_{fu} \geq |\tilde{D}|$, we have:

$$\dot{V} \leq -K_p k_{fu} s^2 - \eta |s|, \quad (5)$$

where $\eta = K_a^* k_{fu} - |\tilde{D}|$ is a positive value, which means that this control law achieve the zero convergence of the sliding surface as time tends to infinity, i.e., $v - v_d \rightarrow 0$ as $t \rightarrow \infty$ [7].

It is worth noting that there may be a realizable issue due to the proposed control law (3), i.e., can we actually acquire $\hat{\omega}_e$ by the measurement of $\dot{\omega}_e$? Thanks to the current technology in filtering, this term $\dot{\omega}_e$ can indeed be obtained through filtering the differenced ω_e by both analog LPF and digital FIR filter coded in DSP chip. It is found that $\hat{\omega}_e$ is a reasonable approximation of $\dot{\omega}_e$ in our case, whereby Eq. (3) can be implemented. Moreover, one may ask a question, i.e., can the sliding-mode controller save the power? Theoretically, it cannot. However, the purpose that we use this controller is to obtain a fast responsive performance and its robustness. So, in order to avoid the chattering that may waste the energy, we will utilize two methodologies to alleviate it. One is to replace the sign function $\operatorname{sgn}(\cdot)$ with a saturation function $\operatorname{sat}(\cdot)$, and the other is to switch off the adaptive law, which is designed for the sliding gain as shown in Eq. (3), when the tracking error is under a critical level. Besides, the drift of the adaptive sliding gain will also be avoided following the modification by means of leakage term [8].

3.2 Energy Management Subsystem Design

When riding an EM, all the power energy comes from the battery, and thus, we develop an energy supervisor to manage all of the events about the energy consumption. We focus on two major subjects. One is the energy indicator, which needs to monitor the state of charge, whereas the other is the regenerative braking operation, which needs to reuse the energy.

Residual capacity estimator: In order to get the initial SOC of the battery, we gauge the open-circuit voltage (OCV) and then utilize the relation between the SOC and the OCV to get the initial value. This method is suitable for static-state but not for dynamic-state estimation due to the battery characteristics. Hence, we then use the Ampere-hour inte-

gration method [9] to calculate the output energy, including the consumed energy Ah_{AHI} outside the battery due to the load, and the dissipated energy Ah_{IR} inside the battery due to the internal resistance which is a function of time. Therefore, the residual capacity can be presented as

$$Ah_{residual} = Ah_{initial} - Ah_{AHI} - Ah_{IR}. \quad (6)$$

However, in order to take the estimation error into account due to the environmental effects, such as the operating temperature, times of usage, age effect, and so on, we prefer to estimate the residual capacity more conservatively to avoid exhausting all the battery energy. So that, we update the initial capacity based on Eq. (7) after a time interval,

$$Ah_{residual,T} = \min\{Ah_{residual,T-1} - Ah_{initial,T(by OCV)}\}, \quad (7)$$

where $Ah_{initial,T(by OCV)}$ is the energy capacity obtained by the OCV method and T stands for the operating period from the EM velocity $v > 0$ to $v = 0$. From Eq. (7), we can see that we update the next initial capacity based on the previous residual capacity we have estimated and the present initial capacity by OCV method. The flowchart for estimating the residual capacity is depicted in Fig. 5.

Regenerative braking system: Theoretically, we can appropriately control the gating signals of the inverter to achieve the regenerative braking. According to a simple but feasible method by the Taiwan UQM Electric Co., as soon as the EM is braking, we then use a fixed duty ratio to be this switching control scheme. However, this fixed duty ratio should be pre-decided according to the ability of the inverter to avoid the damage. Obviously, the regenerative current dynamics is too complex to be derived due to the simple method we have used so that no analytic form is presented here. Even so, we still can obtain the reusable energy Ah_{RG} by the Ampere-hour integration method, in which the “current” term is measured directly. Finally, we have to modify the updated initial capacity as shown in Eq. (8) because of the obtained regenerative energy Ah_{RG} :

$$Ah_{initial,T} = Ah_{initial,T} - Ah_{RG}. \quad (8)$$

3.3. Protection Subsystem Design

In this protection subsystem, we provide three major functionalities, including the over-heat for the BLDCM, over-current for the power driver and the over-discharge protection for the battery. The first two are achieved by the hardware circuit and send the interrupt signals to DSP for warnings. The third one is developed for the batter to avoid the over-use situation, which may cause the damage of the internal structure of the battery. The protection strategies are illustrated in Table II. Notice that obviously the protections do achieve the goal, but it also directly limits the performance of the EM. It immediately imposes effects on the rider’s feeling and then may degrade the speed-following performance of the EM. For this consideration, we decide to leave this protection to be optional according to the rider’s preference.

4. SIMULATION RESULTS AND DISCUSSION

In order to ensure the proposed integrated management

system can achieve the goal, we utilize the MATLAB SIMULINK software to simulate the closed-loop system. Parameters of the EM used here are given by empirical exercises and Taiwan UQM Electric Co.

4.1 Power-Saving Controller

Robust tracking performance: The following case is considered: The EM is grading while the simulation time $t = 30$ sec, $\alpha = 10$ deg., and the total simulation period is 60 sec. The simulation results are shown in Fig. 6. Before the EM is grading, the maximum tracking error is about 0.2 km/hr. However, as soon as climbing starts, the output power rises up rapidly in order to maintain the tracking performance. Although the error has instantly increased, it remains less than 0.8 Km/Hr and converges back to zero in a short time. The robustness of the sliding-mode controller is clearly exhibited.

Power-saving performance: We simulate the EM travelling on an inferior road condition to present the power-saving performance. By the way, in order to illustrate the power-saving capability we’ll use the proposed residual capacity estimator here, so it wouldn’t be illustrated again in the energy management subsystem. As the simulation results in Fig. 7, the slip ratio increases in the beginning due to the inferior road condition. This phenomenon is the so-called “traction control problem”. In this paper, we use the fuzzy controller to reduce this phenomenon and then the unnecessary power loss is saved. As the simulation results shown in Fig. 8, the tracking performance is better than the previous one due to the modification of the fuzzy weighting. Moreover, while the slip ratio is increasing, the corresponding fuzzy weighting falls to modify the output voltage. With this, the contact between the wheel and the road surface is enhanced so that the unnecessary energy loss is avoided. From the simulation result, the saving power is about 0.20%.

4.2. Energy Management Subsystem

First, we have illustrated the residual capacity estimation in the earlier section. Moreover, this simulation result is close to the practical testing result. Thus, we can confirm the correctness of the estimation methodology. Additionally, as mentioned earlier, it is difficult to take the numerical analysis as the regenerative braking operation due to the used methodology. Hence, we only illustrate the residual capacity estimator here.

4.3. Protection Subsystem Design

As discussed before, the over-heat protection for the BLDCM and the over-current protection for the power driver are performed by the hardware circuit. As soon as the dangerous event occurs, an interrupt signal will be sent to the central processing system to execute the corresponding operation, such as setting the output voltage to zero to terminate the in- appropriate motion. Therefore, we only demonstrate the over discharge protection for the battery in the following. While the SOC is higher than 40%, the performance is rarely limited and almost the same with Fig. 6.

Hence, we will not show it again. However, the output voltage is limited while the SOC = 35% and SOC = 15% as the simulations shown above. As we have mentioned before, this protection does protect the useful life of the battery but it obviously limits the performance of the EM. Thus, the protection is optional rather than being inevitable.

5. EXPERIMENTAL RESULTS AND DISCUSSIONS

Here the experiment setup is illustrated and outdoor experimental results are given to verify the validity of the robust velocity-tracking controller design of the EM.

5.1 Experiment Setup

Figure 10 depicts the experiment setup, which has been established in Advanced Control Lab in National Taiwan Univ. Our overall design is implemented by use of TI DSP F240. The driver's command and vehicle velocity are fed to this DSP processor. A portable data collection and online display system are configured, too. We utilize the RS232 unit supported by the EVM board to send the data, such as the driver's command, actual vehicle velocity and so forth, to a laptop computer and display them online by the LabVIEW interface.

5.2 Experimental Result

Figure 11 illustrates the outdoor experimental results tested practically on the road, which is shown in the LabVIEW window, including the driver's input command, actual velocity, tracking error, and output voltage, which are in turn displayed left-to-right and top-to-down. From the labels 1-4 indicated in the experimental result of the tracking error, the larger error is caused because of the driver's accelerating or decelerating command. The major factor that induces the slower velocity tracking response in those areas is the inertia of the EM. Even so, the tracking error will soon converge to almost zero, but maximum error is less than 3 Km/Hr. Practically, there remain some factors that influence the tracking performance, such as the measurement error, mismatch of the utilized parameters, rugged and rough road surface, and so on. Hence, the performance will be absolutely improved if these factors can be considered carefully. Although the velocity tracking is not the overall functionality of the proposed integrated management system, the experimental platform, which we establish here, is indeed a basic and reusable platform for the next-stage development. Based on this setup, more verification will be finished in the future.

6. CONCLUSION

In this paper, we focus on the *highly efficient energy usage* of the battery energy and propose an integrated management system for the EM. This integrated management system includes the power-saving controller, energy management subsystem, and some hardware protection strategies. The power saving controller is designed for the robust velocity tracking control with less unnecessary power dissipation. The energy management system acts as a supervisor to

manage all the events about the battery energy, including the residual capacity estimation and regenerative braking operation. Furthermore, to prevent the damage on the hardware from inappropriate situations, we set up some strategies to avoid them. Numerical simulations are conducted to demonstrate the validity of the performance of the proposed integrated management system. Finally, based on TI DSP F240, we implement our design and verify the performance of the robust velocity tracking control by outdoor experiment. In the future, more performance verification such as the power saving will be achieved.

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Table I Fuzzy rules.

Slip ratio λ	Fuzzy weighting k_{λ}
PB	ZE
PM	PS
ZE	PB
NM	PS
NB	ZE

Table II Protection strategies.

SOC (%)	Max. output voltage
> 40	$V_c < 52$
40~20	$V_c < 30$
< 20	$V_c < 20$

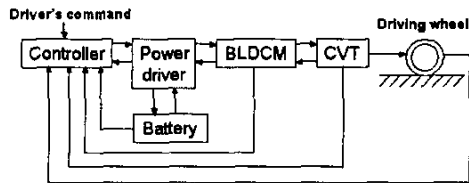


Fig. 1 The architecture diagram of the overall system.

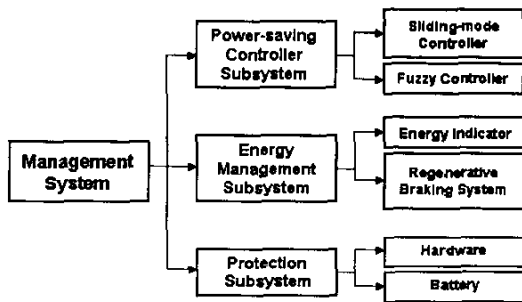


Fig. 2 Hierarchical diagram of the management system.

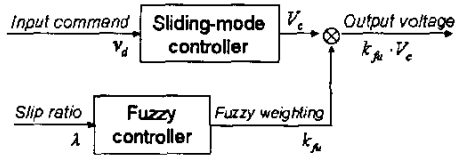


Fig. 3 Power-saving controller architecture.

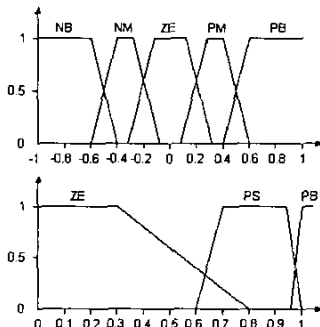


Fig. 4 Membership functions of slip ratio λ (Up) and fuzzy weighting k_{fu} (Down).

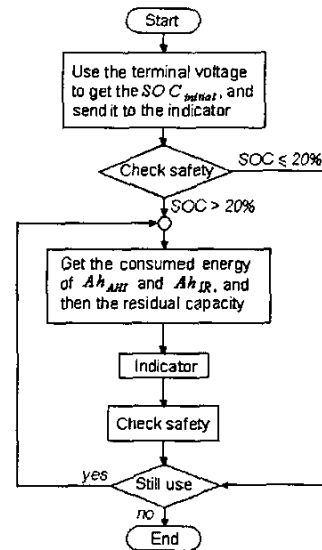
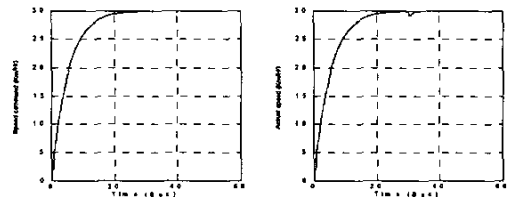
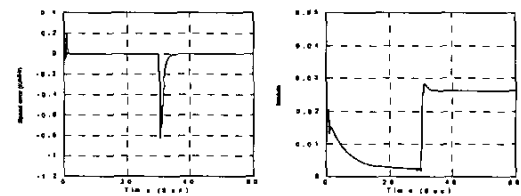


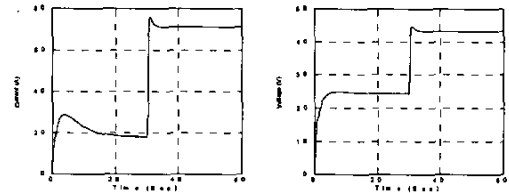
Fig. 5 Flowchart for estimating the residual capacity.



(a) Velocity command [Km/Hr] (b) Actual velocity [Km/Hr]

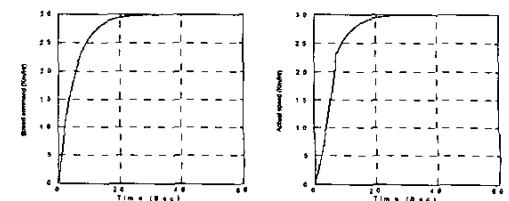


(c) Tracking error [Km/Hr] (d) Slip ratio

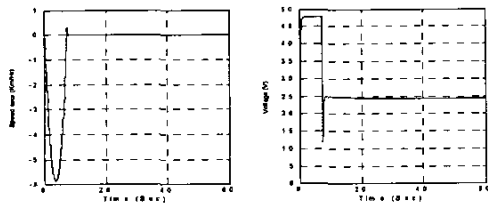


(e) Input current [A] (f) Input voltage [V]

Fig. 6 Climbing performance at $t = 30$ sec.

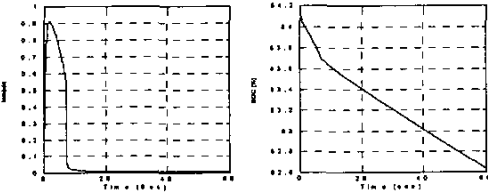


(a) Velocity command [Km/Hr] (b) Actual velocity [Km/Hr]



(c) Tracking error [Km/Hr]

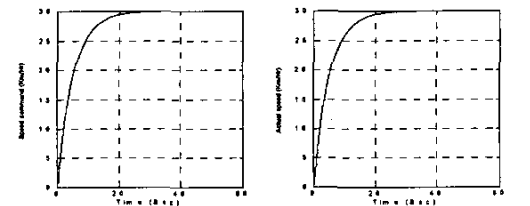
(d) Input voltage [V]



(e) Slip ratio

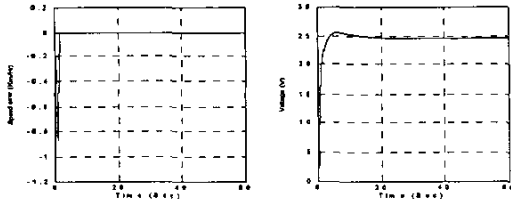
(f) SOC (\downarrow 1.48%)

Fig. 7 Performance without power-saving strategy.



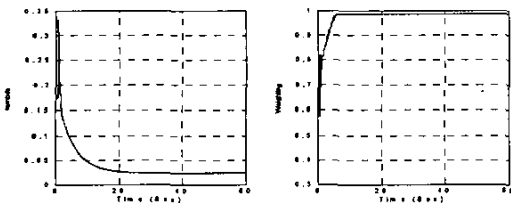
(a) Velocity command [Km/Hr]

(b) Actual velocity [Km/Hr]



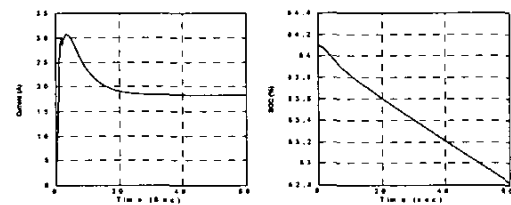
(c) Tracking error [Km/Hr]

(d) Input voltage [V]



(e) Slip ratio

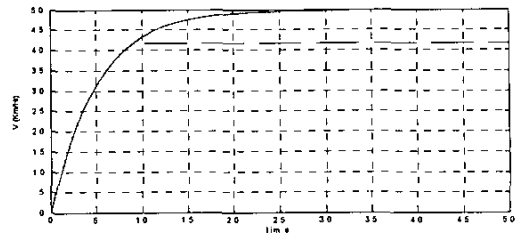
(f) Fuzzy weighting



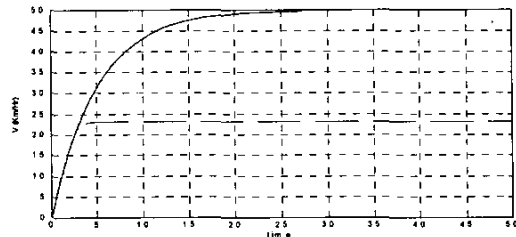
(g) Input current [A]

(h) SOC (\downarrow 1.28%)

Fig. 8 Performance with power-saving strategy.



(a) Velocity response when SOC=35%



(b) Velocity response when SOC=15%

Fig. 9 Protection strategy.

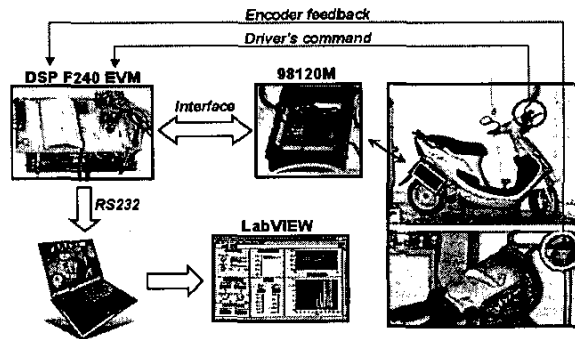


Fig. 10 The overall experimental setting.

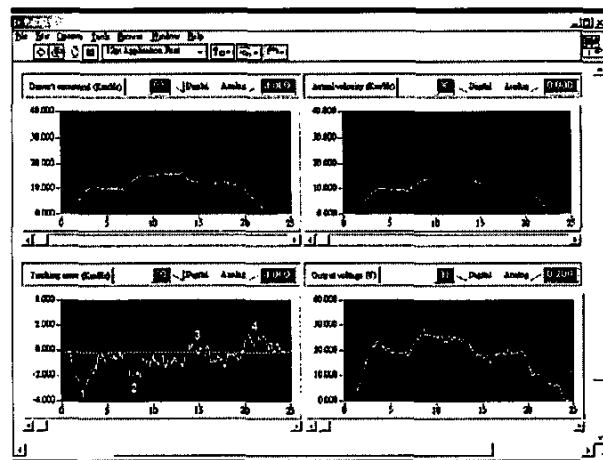


Fig. 11 Experimental results shown by the LabVIEW.